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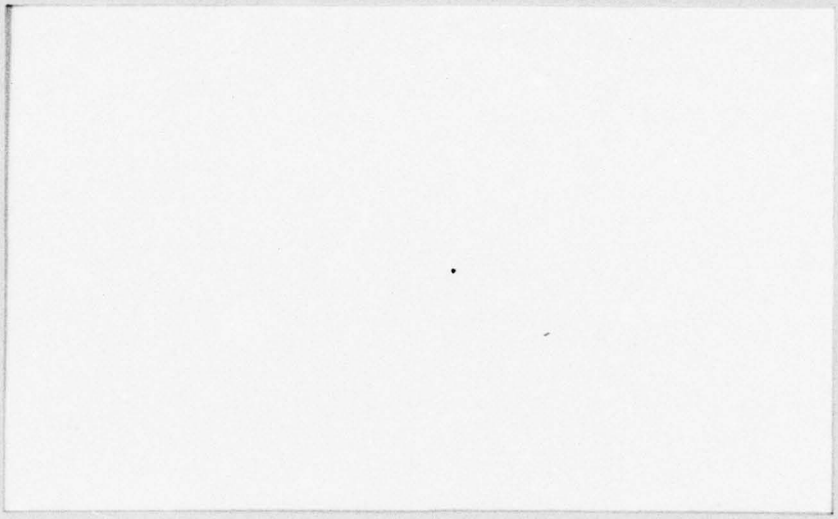

NEW YORK UNIVERSITY
FACULTY OF ARTS AND SCIENCE
DEPARTMENT OF APPLIED SCIENCE

THE EFFECTS OF ACOUSTICAL DISTURBANCES ON BOUNDARY LAYER TRANSITION

G. Miller and A. Callegari

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The analysis of transition from laminar to turbulent flow on a flat plate has been the subject of numerous investigations. In the present work, emphasis has been placed on establishing a computational technique which can be utilized to develop a basic understanding of the effects of the propagation of acoustic waves into a boundary layer, and the ultimate effect of such disturbances on transition. The nonlinear system of unsteady compressible partial differential equations have been solved by a MacCormack predictor-corrector scheme which allows the effects of imposed disturbances to be tracked in time. The question

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of the mismatch in propagation velocity between Tollmein Schlichting and acoustic waves has been studied. The program indicates that while disturbances are propagating with the speed of sound in the inviscid flow, the waves well within the boundary layer are propagating at a speed on the order of the freestream velocity and thus the boundary layer is being excited by the classical Tollmein Schlichting waves. The analysis thus indicates that the effect of acoustical disturbances on transition is similar to the effect of other perturbations as experiments have indicated.

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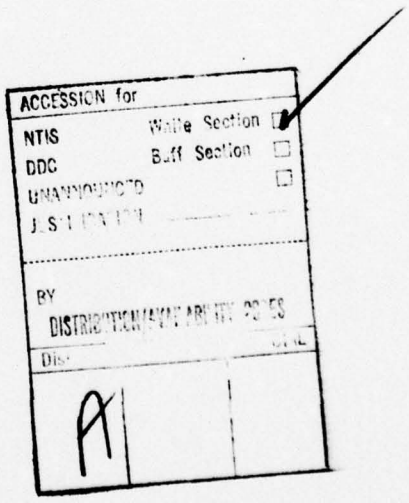

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I. INTRODUCTION

In recent years several investigators (for example, Refs. 1 and 2) have developed numerical analyses to investigate the nonlinear effects of disturbances propagating in a laminar boundary layer with a view towards a better understanding of boundary layer stability. Such investigations are extensions of earlier linear stability analyses (for example, Refs. 3-7) and were motivated by the work of Stuart[8] and others who viewed the transition process as a series of distinctly observable stages. In the first stage, the two dimensional Tollmien-Schlichting waves are developed and the flowfield can be analyzed linearly. However, downstream of this region, nonlinear and three dimensional effects begin to play a more and more dominant role. Thus a complete physical understanding of the transition phenomenon must await the development of a nonlinear, three dimensional program capable of measuring amplification of the waves as they propagate.

Numerical analyses capable of solving the nonlinear system of equations in the boundary layer (even two dimensional codes) can be extremely useful for the study of the wave amplification phenomenon. Linear stability theory, exemplified by eigen solutions of the Orr-Summerfeld equations, cannot directly incorporate the properties of imposed disturbances. Once disturbances are introduced, the mechanism of introduction not appearing in these solutions, it is presumed that they can no longer significantly influence the boundary layer process, which is, perhaps, a serious shortcoming of linear theory.

Nonlinear analyses (such as Refs. 1 and 2) have investigated the stability of laminar, incompressible boundary layers and the initial phase of the transition process (prior to the three dimensional bursting phase) by direct numerical solution of the partial differential equations which describe such a process. The boundary layer flow on a flat plate is disturbed by forced time-dependent

perturbations, and the reaction of the flow, i.e. the temporal and spatial development of the perturbations, is determined by a numerical solution of the governing equations. Such programs thus have the ability to analyze the effects of large scale disturbances (i.e. on the order of .1% of the free stream velocity) and how they are amplified or damped by the boundary layer, whereas linear stability programs cannot directly incorporate the effect of disturbance amplitude into the stability calculation.

While these nonlinear analyses represent a significant advance in the state of the art, they are restricted to incompressible flow. The effects of such parameters as Mach number and wall temperature are, of course, of great significance and a compressible analog to these methods is necessary.

In addition, such programs are unable to determine directly the effects of imposed acoustical disturbances. Experiments, for example Refs. 9 and 10, have indicated that the effects of imposed acoustic perturbations on boundary layer transition is similar to the effects of freestream turbulence[11] in low speed flow. One would expect waves propagating through the flowfield at the speed of sound to affect boundary layer transition in a different way than waves propagating through at the freestream velocity, but the Spangler and Wells experiments, as well as others, have shown that this is not the case. The frequency range which affects transition at a given Reynolds number is the same as when freestream disturbances are introduced, even though there appears to be a mismatch in propagation velocity (and thus in wave number).

The necessity for a study of the effects of these acoustical disturbances was the motivation of the present investigation and it's extension. The unsteady, compressible,second order boundary layer equations have been utilized. Terms of the order of the reciprocal of the Reynolds number and the reciprocal squared have been retained and thus the system is consistent with the Navier Stokes equations. These equations have been solved by an

explicit scheme in the high Reynolds number range (where the equations are, essentially, the boundary layer equations) consistent with the experiments of Spangler and Wells.

In Section II, the equations of motion and, method of analysis is developed. In Section III, the results and conclusions of the present investigation are presented. This work is part of a continuing effort and additional results, especially with regards to the effects of such parameters as Mach number and wall temperature, will be forthcoming.

II. METHOD OF ANALYSIS

A. Equations of Motion

The system of equations have have been utilized are the compressible, two-dimensional, higher order boundary layer equations (retaining terms of order $1/Re$ and $1/Re^2$). This system is thus, essentially, the compressible Navier Stokes equations. For unit Prandtl number, the system, for a perfect gas is;

$$\frac{\partial \bar{\rho}}{\partial \bar{t}} + \frac{\partial}{\partial \bar{x}} (\bar{\rho}\bar{u}) + \frac{\partial}{\partial \bar{y}} (\bar{\rho}\bar{v}) = 0 \tag{1}$$

$$\frac{\partial}{\partial \bar{t}} (\bar{\rho}\bar{u}) + \frac{\partial}{\partial \bar{x}} (\bar{\rho}\bar{u}^2 + \bar{p}) + \frac{\partial}{\partial \bar{y}} (\bar{\rho}\bar{u}\bar{v}) = \frac{\partial}{\partial \bar{y}} [\bar{\mu} \frac{\partial \bar{u}}{\partial \bar{y}}] + \phi_1$$

$$\frac{\partial}{\partial \bar{t}} (\bar{\rho}\bar{v}) + \frac{\partial}{\partial \bar{x}} (\bar{\rho}\bar{u}\bar{v}) + \frac{\partial}{\partial \bar{y}} [\bar{\rho}\bar{v}^2 + Re_L \cdot \bar{p}] = \frac{4}{3} \frac{\partial}{\partial \bar{y}} [\bar{\mu} \frac{\partial \bar{v}}{\partial \bar{y}}] \tag{2}$$

$$- \frac{2}{3} \frac{\partial}{\partial \bar{y}} (\bar{\mu} \frac{\partial \bar{u}}{\partial \bar{x}}) + \frac{\partial}{\partial \bar{x}} (\bar{\mu} \frac{\partial \bar{u}}{\partial \bar{y}}) + \phi_2 \tag{3}$$

$$\frac{\partial}{\partial \bar{t}} (\bar{\rho}\bar{E}) + \frac{\partial}{\partial \bar{x}} (\bar{\rho}\bar{u}\bar{H}) + \frac{\partial}{\partial \bar{y}} (\bar{\rho}\bar{v}\bar{H}) = \frac{\partial}{\partial \bar{y}} [\bar{\mu} \frac{\partial \bar{H}}{\partial \bar{y}}] + \phi_3 \tag{4}$$

$$\bar{p} = \bar{\rho}\bar{T} \tag{5}$$

where

$$\bar{x} = \frac{x}{L}; \quad \bar{y} = \frac{y}{L} \cdot \sqrt{Re_L}; \quad \bar{t} = t \frac{u_\infty}{L}$$

$$\bar{u} = \frac{u}{u_\infty}; \quad \bar{v} = \frac{v}{u_\infty} \cdot \sqrt{Re_L}; \quad \bar{p} = \frac{p}{\rho_\infty u_\infty^2}; \quad \bar{\rho} = \frac{\rho}{\rho_\infty}$$

$$\bar{H} = \frac{H}{u_\infty^2} = \frac{C_p T}{u_\infty^2} + \frac{u^2 + v^2}{2u_\infty^2}; \quad \bar{E} = \frac{E}{u_\infty^2} = \frac{C_v T}{u_\infty^2} + \frac{u^2 + v^2}{2u_\infty^2}$$

$$\bar{\mu} = \frac{\mu}{\mu_\infty} = \frac{T_\infty + 198.6}{T + 198.6} \left(\frac{T}{T_\infty}\right)^{3/2}; \quad \bar{T} = \frac{RT}{u_\infty^2} \qquad Re_L = \frac{\rho_\infty u_\infty L}{\mu_\infty}$$

$$\phi_1 = \frac{1}{Re_L} [\frac{4}{3} \frac{\partial}{\partial \bar{x}} (\bar{\mu} \frac{\partial \bar{u}}{\partial \bar{x}}) - \frac{2}{3} \frac{\partial}{\partial \bar{x}} (\bar{\mu} \frac{\partial \bar{v}}{\partial \bar{y}}) + \frac{\partial}{\partial \bar{y}} (\bar{\mu} \frac{\partial \bar{v}}{\partial \bar{x}})]$$

$$\phi_2 = \frac{1}{Re_L} \frac{\partial}{\partial \bar{x}} (\bar{\mu} \frac{\partial v}{\partial \bar{x}})$$

$$\phi_3 = \frac{1}{Re_L}\left[\frac{\partial}{\partial\bar{x}}\left(\bar{\mu}\,\frac{\partial\bar{H}}{\partial\bar{x}}\right) + \frac{1}{3}\,\bar{\mu}\left(\frac{\partial\bar{u}}{\partial\bar{x}}\right)^2 + \frac{1}{3}\,\bar{\mu}\left(\frac{\partial\bar{v}}{\partial\bar{y}}\right)^2 - \frac{2}{3}\bar{\mu}\,\frac{\partial\bar{u}}{\partial\bar{x}}\,\frac{\partial\bar{v}}{\partial\bar{y}} + 2\bar{\mu}\,\frac{\partial\bar{v}}{\partial\bar{x}}\,\frac{\partial\bar{u}}{\partial\bar{y}}\right.$$

$$\left. - \frac{1}{2}\,\frac{\partial\bar{\mu}}{\partial\bar{x}}\,\frac{\partial}{\partial\bar{x}}\left(\bar{u}^2\right) - \frac{1}{2}\,\frac{\partial\bar{\mu}}{\partial\bar{x}}\,\frac{\partial\bar{v}^2}{\partial\bar{x}}\,\frac{1}{Re_L} - \frac{1}{2}\,\frac{\partial\bar{\mu}}{\partial\bar{y}}\,\frac{\partial\bar{v}^2}{\partial\bar{y}}\right]$$

$$- \frac{1}{2}\,\frac{\partial\bar{\mu}}{\partial\bar{y}}\,\frac{\partial\bar{u}^2}{\partial\bar{y}}$$

In order to study the effect of the propagation of acoustical free-stream disturbances, the equations have been transformed by a stretched coordinate system which packs many points into the boundary layer, but allows for a significant distribution of data in the freestream. The transformation utilized is:

$$\eta = 1 - \exp(-\alpha\bar{y}/\bar{\delta}^*) \tag{6}$$

where $\bar{\delta}^* = \delta_o^*/L\sqrt{Re_L}$ and $\delta_o^*(x)$ is the initial boundary layer displacement thickness distribution (i.e., the distribution at $\bar{t} = 0$). Thus by varying α one can pack as many points as needed near the surface (note as $\bar{y} \to \infty, \eta \to 1$).

The system of equations in the transformed system is;

$$\frac{\partial\bar{\rho}}{\partial\bar{t}} + \frac{\partial}{\partial\bar{x}}(\bar{\rho}\bar{u}) + \frac{\partial}{\partial\eta}\,(\bar{\rho}\bar{u})\,\frac{\partial\eta}{\partial\bar{x}} + (1-\eta)\,\frac{\alpha}{\bar{\delta}^*}\,\frac{\partial\bar{\rho}\bar{v}}{\partial\eta} = 0 \tag{7}$$

$$\frac{\partial}{\partial\bar{t}}\,(\bar{\rho}\bar{u}) + \frac{\partial}{\partial\bar{x}}\,(\bar{\rho}\bar{u}^2 + \bar{p}) + \frac{\partial}{\partial\eta}\,(\bar{\rho}\bar{u}^2 + \bar{p})\,\frac{\partial\eta}{\partial\bar{x}} + (1-\eta)\,\frac{\alpha}{\bar{\delta}^*}\,\frac{\partial}{\partial\eta}\,(\bar{\rho}\bar{u}\bar{v})$$

$$= \frac{\alpha^2(1-\eta)}{\bar{\delta}^{*2}}\,\frac{\partial}{\partial\eta}\,\left[\bar{\mu}\,(1-\eta)\,\frac{\partial\bar{u}}{\partial\eta}\right] + \phi_4 \tag{8}$$

$$\frac{\partial}{\partial \bar{t}}(\bar{\rho}\bar{v}) + \frac{\partial}{\partial \bar{x}}(\bar{\rho}\bar{u}\bar{v}) + \frac{\partial}{\partial \eta}(\bar{\rho}\bar{v}\bar{v})\frac{\partial \eta}{\partial \bar{x}} - (1-\eta)\frac{\alpha}{\bar{\delta}^*}\frac{\partial}{\partial \eta}(\bar{\rho}\bar{v}^2 + \bar{p}\ Re_L) \tag{9}$$

$$= \frac{4}{3}\frac{\alpha^2}{\bar{\delta}^{*2}}(1-\eta)\frac{\partial}{\partial \eta}[\bar{\mu}(1-\eta)\frac{\partial \bar{v}}{\partial \eta}] - \frac{2}{3}(1-\eta)\frac{\alpha}{\bar{\delta}^*}\frac{\partial}{\partial \eta}[\bar{\mu}\frac{\partial \bar{u}}{\partial \bar{x}} + \bar{\mu}\frac{\partial \eta}{\partial \bar{x}}\frac{\partial \bar{u}}{\partial \eta}]$$

$$+ \frac{\partial}{\partial \bar{x}}[\bar{\mu}(1-\eta)\frac{\alpha}{\bar{\delta}^*}\frac{\partial \bar{u}}{\partial \eta}] + \frac{\partial \eta}{\partial \bar{x}}\frac{\partial}{\partial \eta}[\bar{\mu}(1-\eta)\frac{\alpha}{\bar{\delta}^*}\frac{\partial \bar{u}}{\partial \eta}] + \phi_5$$

$$\frac{\partial}{\partial \bar{t}}(\bar{\rho}\bar{E}) + \frac{\partial}{\partial \bar{x}}(\bar{\rho}\bar{u}\bar{H}) + \frac{\partial}{\partial \eta}(\bar{\rho}\bar{u}\bar{H})\frac{\partial \eta}{\partial \bar{x}} + (1-\eta)\frac{\alpha}{\bar{\delta}^*}\frac{\partial}{\partial \eta}(\bar{\rho}\bar{u}\bar{H}) \tag{10}$$

$$= \frac{\alpha^2(1-\eta)}{\bar{\delta}^{*2}}\frac{\partial}{\partial \eta}[\bar{\mu}(1-\eta)\frac{\partial \bar{H}}{\partial \eta}] + \phi_6$$

$$\bar{p} = \bar{\rho}\bar{T} \tag{11}$$

where

$$\phi_4 = \frac{1}{Re_L}\{\frac{4}{3}\frac{\partial}{\partial \bar{x}}(\bar{\mu}\frac{\partial \bar{u}}{\partial \bar{x}} - \frac{2}{3}\frac{\alpha}{\bar{\delta}^*}\frac{\partial}{\partial \bar{x}}[\bar{\mu}(1-\eta)\frac{\partial \bar{v}}{\partial \eta}]$$

$$+ \frac{4}{3}\frac{\partial \eta}{\partial \bar{x}}\frac{\partial}{\partial \eta}(\bar{\mu}\frac{\partial \bar{u}}{\partial \bar{x}}) - \frac{2}{3}\frac{\alpha}{\bar{\delta}^*}\frac{\partial \eta}{\partial \bar{x}}\frac{\partial}{\partial \eta}[\bar{\mu}(1-\eta)\frac{\partial \bar{v}}{\partial \eta}]$$

$$+ (1-\eta)\frac{\alpha}{\bar{\delta}^*}\frac{\partial}{\partial \eta}(\bar{\mu}\frac{\partial \bar{v}}{\partial \bar{x}})$$

$$+ \frac{4}{3}\frac{\partial}{\partial \bar{x}}(\bar{\mu}\frac{\partial \eta}{\partial \bar{x}}\frac{\partial \bar{u}}{\partial \eta}) + \frac{4}{3}\frac{\partial \eta}{\partial \bar{x}}\frac{\partial}{\partial \eta}(\bar{\mu}\frac{\partial \eta}{\partial \bar{x}}\frac{\partial \bar{u}}{\partial \eta})$$

$$+ \frac{\alpha}{\bar{\delta}^*}(1-\eta)\frac{\partial}{\partial \eta}(\bar{\mu}\frac{\partial \eta}{\partial \bar{x}}\frac{\partial \bar{v}}{\partial \eta})\}$$

$$\phi_5 = \frac{1}{Re_L}[\frac{\partial}{\partial \bar{x}}(\bar{\mu}\frac{\partial \bar{v}}{\partial \bar{x}}) + \frac{\partial \eta}{\partial \bar{x}}\frac{\partial}{\partial \eta}(\bar{\mu}\frac{\partial \bar{v}}{\partial \bar{x}})$$

$$+ \frac{\partial}{\partial \bar{x}}(\frac{\partial \eta}{\partial \bar{x}}\frac{\partial \bar{v}}{\partial \eta}) + (\frac{\partial \eta}{\partial \bar{x}})\frac{\partial}{\partial \eta}(\frac{\partial \eta}{\partial \bar{x}}\frac{\partial \bar{v}}{\partial \eta})]$$

$$\phi_6 = \frac{1}{Re_L}\{[\frac{\partial}{\partial \bar{x}} + \frac{\partial \eta}{\partial \bar{x}}\frac{\partial}{\partial \eta}][\bar{\mu}\frac{\partial \bar{H}}{\partial \bar{x}} + \frac{\partial \eta}{\partial \bar{x}}\frac{\partial \bar{H}}{\partial \eta}]$$

$$+ \frac{1}{3}\bar{\mu}(\frac{\partial \bar{u}}{\partial \bar{x}} + \frac{\partial \eta}{\partial \bar{x}}\frac{\partial \bar{u}}{\partial \eta})^2 + \frac{1}{3}\bar{\mu}(\frac{\partial \bar{v}}{\partial \bar{x}} + \frac{\partial \eta}{\partial \bar{x}}\frac{\partial \bar{v}}{\partial \eta})^2$$

$$- \frac{2}{3}\frac{\alpha(1-\eta)}{\bar{\delta}^*}\bar{\mu}(\frac{\partial \bar{u}}{\partial \bar{x}} + \frac{\partial \eta}{\partial \bar{x}}\frac{\partial \bar{u}}{\partial \eta})\frac{\partial \bar{v}}{\partial \eta}$$

$$+ \frac{2\alpha(1-\eta)}{\bar{\delta}^*}\bar{\mu}(\frac{\partial \bar{v}}{\partial \bar{x}} + \frac{\partial \eta}{\partial \bar{x}}\frac{\partial \bar{v}}{\partial \eta})\frac{\partial \bar{u}}{\partial \eta}$$

$$- \frac{1}{2}(\frac{\partial \bar{u}}{\partial \bar{x}} + \frac{\partial \eta}{\partial \bar{x}}\frac{\partial \bar{u}}{\partial \eta})(\frac{\partial(\bar{u}^2)}{\partial \bar{x}} + \frac{\partial \eta}{\partial \bar{x}}\frac{\partial \bar{u}^2}{\partial \eta})$$

$$- \frac{1}{2Re_L} \left(\frac{\partial \bar{\mu}}{\partial \bar{x}} + \frac{\partial \eta}{\partial \bar{x}} \frac{\partial \bar{\mu}}{\partial \eta} \right) \left(\frac{\partial \bar{v}^2}{\partial \bar{x}} + \frac{\partial \eta}{\partial \bar{x}} \frac{\partial \bar{v}^2}{\partial \eta} \right)$$

$$- \frac{1}{2} \frac{\alpha^2}{\bar{\delta}^{*2}} (1 - \eta)^2 \frac{\partial \bar{v}^2}{\partial \eta} \Bigg\}$$

$$- \frac{\alpha^2}{\bar{\delta}^{*2}} (1 - \eta)^2 \frac{\partial \bar{\mu}}{\partial \eta} \frac{\partial \bar{u}^2}{\partial \eta}$$

Equations (7) through (11) are the basic system of equations which have been numerically solved.

B. Initial and Boundary Conditions

The investigation has centered on a study of the amplification or damping of acoustical disturbances propagated into a boundary layer. The experiments of Spangler and Wells[9] who utilized an air-driven, rotating vane sound generator to create the disturbance without producing any appreciable turbulence, were used. Both the frequency and intensity of the sound source were varied experimentally. A low velocity boundary layer channel was run at a unit Reynolds number of 2.4×10^5/ft., the channel wall representing the flat plate. Measurements of transition occurred at distances on the order of 10-20 feet, and thus the length Reynolds numbers of interest are in the 10^6 to 10^7 range (see next section).

In order to model this problem, a set of initial and boundary conditions must be established, consistent with the propagation of acoustical disturbances and at the same time consistent with the set of differential equations utilized. The initial data, consistent with equations (1) through (5) under steady state conditions, was originally established utilizing numerical techniques for subsonic boundary layer analysis with normal pressure gradients established by

the principal investigator and reported previously, i.e., Ref. 12. Thus the solution at $\bar{t} = o$ was the solution of the two dimensional steady boundary layer equations with normal momentum equation included.

During the past year, consistent with Refs. 1 and 2, the Blasius solution has been utilized (since for the cases of interest, $M_\infty \approx .03$) and has caused no major problems.

At the wall ($\eta = o$), for a rigid body, one can neglect the effect of the wave on temperature, and for an adiabatic wall one can establish the following relations (if one drops terms of order 1/Re which implies some inconsistency for lower Re)

$$\eta = 0 \quad (\bar{x} > \bar{x}^*)$$

$$\bar{u} = \bar{v} = \frac{\partial \bar{T}}{\partial \eta} = 0$$

$$\frac{\partial}{\partial \eta}(\ell n \bar{\rho}) = \frac{\partial}{\partial \eta}[(\ell n \bar{\rho})]_{\bar{t}=0} \exp\left[-Re_L \int_o^{\bar{t}} \frac{\bar{p}}{\bar{\mu}} \, d\bar{t}\right]$$

$$\frac{\partial}{\partial \eta}(\ell n \bar{\rho}) = \frac{\partial}{\partial \eta}[(\ell n \bar{\rho})]_{\bar{t}=0} \exp\left[-Re_L \int_o^{t} \frac{\bar{p}}{\bar{\mu}} d\bar{t}\right]$$

The outer boundary condition ($\eta \to 1$) is established by allowing the wave to travel as a plane wave with speed of sound c_∞.

Much effort has been placed on the determination of a proper downstream boundary condition. It has been found, consistent with the results of Fasel[1], that the boundary condition that yields the least upstream influence, and is thus superior to other possible ones (including a non reflective condition) is:

$$\frac{\partial^2 \bar{u}'(\bar{x}_f, \bar{y}, \bar{t})}{\partial \bar{x}^2} = \bar{\alpha}^2 \bar{u}'$$

where $\bar{\alpha} = fL/V_{ph}$ (f is the frequency and V_{ph} the phase velocity), $\bar{u}'$ is the perturbation quantity ($\bar{u} - \bar{u}_o$), and $\bar{x}_f$ is the downstream boundary. This condition says that at the downstream boundary, the disturbance has a periodic form.

At $\bar{t} = o$, an acoustical disturbance is initiated at $\bar{x} \approx o$ so that the velocity field at $x = x^* \approx o$ and $\bar{t} > o$ can be written as

$$\bar{u}(\bar{x}^*, \bar{y}, \bar{t}) = \bar{u}_o(\bar{x}^*, \bar{y}) + f_1(\bar{y}) \sin(\bar{\omega}\bar{t})$$

where $\bar{u}_o(\bar{x}^*, \bar{y})$ is the velocity profile at $\bar{t} = o$, $\bar{\omega} = 2\pi fL/U_\infty$ (f being the frequency of the imposed disturbance, in cycles per second) and $f_1(\bar{y})$ is the disturbance profile near the leading edge. The form of $f_1(\bar{y})$ has been the object of extensive analysis and is described here. The other boundary conditions at $x = x^*$ are consistent with linear theory and are determined once the form of $f_1(\bar{y})$ is known.

In order to derive appropriate perturbation profiles for use as the initial boundary condition in our program (at $\bar{x} = \bar{x}^*$) the compressible analogue of the Orr-Sommerfeld equations must be solved. We have at present considered these equations in the inviscid limit and under the additional assumption of no temperature gradient in the mean flow.

Thus nondimensionalizing all lengths by the boundary layer thickness δ at a suitable station along the plate, all velocities by the free stream speed of sound, and the pressure by $\rho_\infty c_\infty^2$, the linearized perturbation system solved is;

$$i(ku_o - \omega)\rho + iku\rho_o + \frac{d}{dy}(\rho_o V) = 0 \tag{12}$$

$$i(ku_o - \omega)u + u_o' V + \frac{ik}{\rho_o} p = o \tag{13}$$

$$i(ku_o - \omega)V + \frac{1}{\rho_o}\frac{dp}{dy} = o \tag{14}$$

$$i(ku_o - \omega)T + T_o' V = i\left(\frac{\gamma - 1}{\rho_o}\right)(ku_o - \omega)p \tag{15}$$

$$p/p_o = \rho/\rho_o + T/T_o \tag{16}$$

where bars have been dropped and subscript zero refers to the profile at $t = o$. The appropriate boundary conditions are discussed below. As is customary the normal mode decomposition has been used, and hence we note that ρ, u, V, p, T

are functions of y, the distance normal to the plate. The parameters ω and k, the dimensionless frequency and wave number respectively are given by

$$\omega = \Omega\delta/c_\infty, \quad k = \bar{k}\delta$$

where Ω and $\bar{k}$ are dimensional quantities. The mean flow quantities

$$u_o = M_\infty u_B \text{ and } u_o' = 5.6\ (M_\infty \frac{du_B}{d\eta})$$

are the dimensionless velocity and velocity gradient respectively where M_∞ is the freestream Mach number, u_B is the dimensionless Blasius profile and the factor 5.6 results from changing the normal coordinate from η to y.

Several simplifications can be made in equations (12 - 16). We have solved for ρ, u, v and T in terms of the pressure p and find that p must satisfy the second order equation

$$p'' - \frac{2ku_o'}{u_o k-\omega} p' + \left[\frac{(u_o k - \omega)^2}{T_o} - k^2 \right] p = o \qquad (17)$$

In solving this equation we have considered ω as fixed and k as an unknown to be found. (In general, the phase velocity, $V_p = \omega/k$ and the wave number can be complex). For the present we are considering neutral stability and thus k and V_p are real, however, we plan to extend the calculation to include complex k.

To derive suitable boundary conditions, we assume that at the plate the perturbation V, in the normal velocity is zero and since

$$V = \frac{i}{\rho_o(ku_o - \omega)} \frac{dP}{dy}$$

we take

$$\frac{dP}{dy} = o \text{ at } y = o.$$

To derive a boundary condition at the top of the boundary layer or in the free stream, we write equation (17) in the free stream, i.e. with $u_o' = o$ and $u_B = 1$. Then the equation has constant coefficients and can be readily solved. The nature of the solutions, depends of course, on the sign of $\lambda = -(M_\infty k - \omega)^2 + k^2$.

For the frequency and Mach numbers considered, λ is positive and hence $p = C_1 \exp(-\sqrt{\lambda}y)$ is the solution of (17) which vanishes at infinite distances from the plate. Using this solution to provide boundary conditions for the top of the boundary layer we have

$$\frac{dp}{dy}(1) = -\sqrt{\lambda} = -\sqrt{k^2 - (M_\infty k - \omega)^2} \quad \text{and } p(1) = 1 \tag{18}$$

We note that the second condition is a normalization condition.

Equations (12) - (17) now provide an eigenvalue problem for the eigenvalue k. Once k is determined, the profiles at $x = \bar{x}^*$ (i.e. the $f_1(y)$) can be determined and thus the initial and boundary conditions for the calculation are entirely prescribed.

C. Numerical Method

The MacCormack predictor corrector scheme has been utilized in the study. Equations (7) - (10) are four equations of the form

$$\frac{\partial e}{\partial t} = a_1 \frac{\partial f}{\partial x} + a_2 \frac{\partial g}{\partial \eta} + a_3 \frac{\partial^2 h}{\partial \eta^2} + a_4 \frac{\partial^2 i}{\partial \bar{x}^2} + a_5 \frac{\partial^2 j}{\partial \bar{x} \partial \eta}$$

If these equations are written as

$$\frac{\partial e_i}{\partial t} (\bar{x}, \eta, \bar{t}) = F(\bar{x}, \eta, \bar{t})$$

$$e_1 = \bar{\rho}, \quad e_2 = \bar{\rho}\bar{u}, \quad e_3 = \bar{\rho}\bar{v}, \quad e_4 = \bar{\rho}\bar{E}$$

the MacCormack method states that

$$e(\bar{x}, \eta, \bar{t} + \Delta\bar{t}) = e(\bar{x}, \eta, \bar{t}) + 1/2[F(\bar{x}, \eta, \bar{t}) + F(\bar{x}, \eta, \bar{t} + \Delta\bar{t})]\Delta\bar{t}$$

In order to solve for $e(\bar{x}, \eta, \bar{t} + \Delta\bar{t})$ one first calculates a provisional solution,

$$e^{(1)}(\bar{x}, \eta, \bar{t} + \Delta\bar{t}) = e(\bar{x}, \eta, \bar{t}) + F^{(+)}(\bar{x}, \eta, \bar{t})\, \Delta\bar{t}$$

where $F^{(+)}$ refers to evaluation by non-centered differences, using values at $\bar{x} + \Delta\bar{x}$, $\eta + \Delta\eta$, $\bar{x}$, and η, e. g.

$$\frac{\partial f}{\partial x} = \frac{f(\bar{x} + \Delta\bar{x}, \eta, \bar{t}) - f(\bar{x}, \eta, \bar{t})}{\Delta\bar{x}}$$

then one can use $e^{(1)}$ to evaluate the term $F^{(-)}(\bar{x}, \eta, \bar{t} + \Delta\bar{t})$ using non-centered differences based on $\bar{x} - \Delta\bar{x}$, $\eta - \Delta\eta$, $\bar{x}$, η; e.g.,

$$\frac{\partial f}{\partial x} = \frac{f^{(-)}(\bar{x}, \eta, \bar{t} + \Delta\bar{t}) - f^{(-)}(\bar{x} - \Delta\bar{x}, \eta, \bar{t} + \Delta\bar{t})}{\Delta\bar{x}}$$

Thus one can solve for $e(\bar{x}, \eta, \bar{t} + \Delta\bar{t})$ from the system,

$$e(\bar{x}, \eta, \bar{t} + \Delta\bar{t}) = 1/2[e(\bar{x}, \eta, \bar{t}) + e^{(1)}(\bar{x}, \eta, \bar{t} + \Delta\bar{t}) + F^{(-)}(\bar{x}, \eta, \bar{t} + \Delta\bar{t})\, \Delta\bar{t}]$$

Equations (7) - (10), the full unsteady compressible Navier Stokes system are a complex set of elliptic differential equations to numerically integrate. No other nonlinear analysis, that we are aware of, has attempted to solve the compressible system for the present application (both Fasel and Murdock solve the incompressible problem). The explicit MacCormack scheme was seen as a simpler method for the solution of this system as opposed to an implicit (or ADI) method.

The problem with utilizing an explicit scheme for such calculations is manifested in the problem of the time step which can be utilized in order to insure a stable solution. Since $\Delta y << \Delta x$ in the boundary layer, the time step to satisfy the C.F.L. condition is such that $\Delta t < \Delta y/c$, which is extremely small, and in order to insure on the order of ten time periods for the frequencies of interest, on the order of 10^5 time steps would be required. We have noticed though that for the problems of interest (the modeling of the experiments of Ref. 8) the local Reynolds numbers of interest are greater than 10^6 and thus the leading terms are the parabolic system (i.e., the first order boundary layer equations). The parabolic system has a much less stringent time size criteria (like $\Delta t < \Delta y^2/2\nu$).

For the grid sizes used in the present investigation, $\Delta\bar{t} \sim M_\infty \Delta\bar{x}$ satisfies the parabolic time step criteria ($\Delta\bar{t} < \Delta\bar{y}^2/2$). Over the time scale of interest (on the order of ten perturbation periods) the relaxation of the C.F.L. criteria is manifested not in a gross instability (because of the very high Reynolds numbers) but as a drift in the solution. This was noticed by imposing a zero perturbation field at the station $\bar{x} = \bar{x}^*$ (i.e., $f_1(y) = 0$) and tracking the perturbation solution ($\bar{u} - \bar{u}_{Blasius}$). The program was thus modified to circumvent this problem in the following manner. At any time step, the equations are solved under two sets of conditions, first for the flow without external disturbance (i.e., $f_1(y) = 0$) and then for the flow with disturbance. The solution of every point at each time step is then taken as

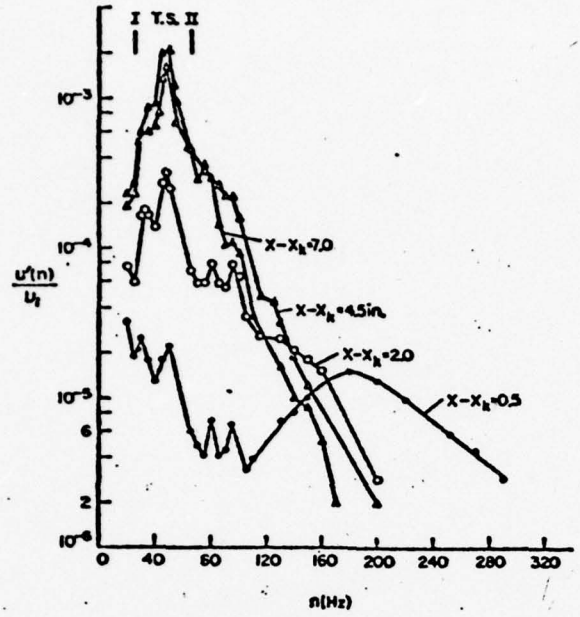
the difference between the two solutions. Thus it is felt that any errors due to numerics have been suppressed. The time traces which will be presented in the next section exhibit no drift and, thus, we feel that the oscillations at each position in the flowfield are due only to the disturbance imposed and are not numerical in nature. At lower Reynolds numbers or for times greater than those of the present calculations the error introduced by not satisfying the C.F.L. will grow and an instability will undoubtedly occur, necessitating one to resort to time splitting (or some other such technique). During the coming year, the program will be modified towards this end.

IV. THE PRESENT INVESTIGATION AND RESULTS TO DATE

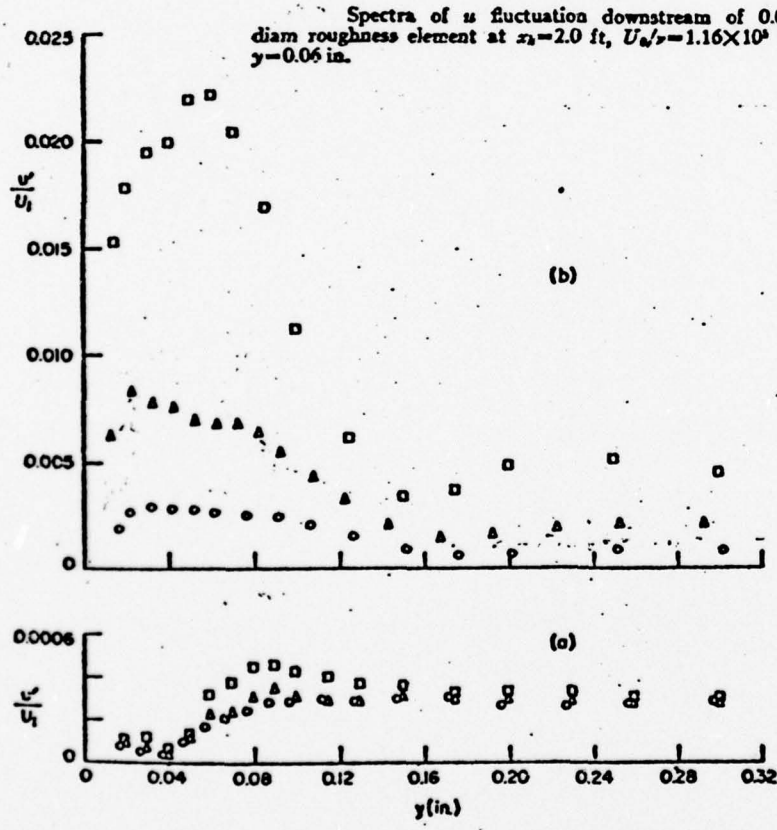
One of the tasks that was pursued this year was an attempt to interpret the results of experimental investigations of large scale (.1% to 1% of freestream) perturbations on boundary layer stability. The experimental results of Klebanoff and Tidstrom [14] for boundary layer transition induced by a two dimensional roughness element were examined in detail towards this end.

Figure 1 (reproduced from Reference 14) presents the longtitudinal velocity fluctuations downstream of the roughness element. In the top figure, the fluctuations directly downstream of the element (located at $x = x_k = 2$ ft) for a unit Reynolds number of 1.16×10^5/ft are plotted as a function of frequency. At the end of the recovery zone ($x - x_k \approx 4.5$ in) amplification in the Tollmien Schlichting range has occurred. In the lower figure the transverse profiles, for a range of unit Reynolds numbers are presented. At larger unit Reynolds numbers, transition occurred in the recovery zone and the analysis presented below is no longer pertinent. For the unit Reynolds number 1.16×10^5/ft the boundary layer profile became highly inflected (and more unstable) downstream of the element until $x - x_k = 4.5$ in where the profile returns to a Blasius representation, as is shown in Figure 2 taken from Reference 14. For this case, transition occurs at $x - x_k = 19$ in.

Thus for a length of approximately two feet, a Blasius like flowfield is under the influence of a large perturbation ($U'/U_\infty > .1\%$ at $x - x_k = 4.5$ in) initial profile. This initial profile is shown by the triangles in (b) of the lower figure of Figure 1. This profile corresponds to a linear spatial stability profile for 50 cycles/sec (the dominant mode for the case) but for a somewhat higher Reynolds number (the roughness element has, of course, amplified the

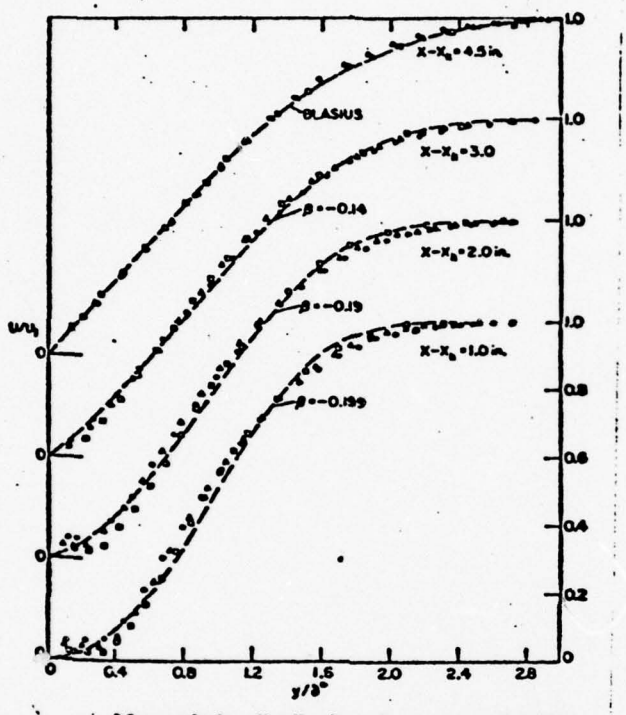


Spectra of u fluctuation downstream of 0.066-in. diam roughness element at x_k=2.0 ft, U_e/ν=1.16×10^5 (ft^{-1}), y=0.06 in.



Intensity of u fluctuation downstream of 0.066-in. diam roughness element at x_k=2.0 ft, illustrating increase in amplification within recovery zone. ⊙ U_e/ν=1.0×10^5 (ft^{-1}), △ U_e/ν=1.16×10^5 (ft^{-1}), ⊡ U_e/ν=1.42×10^5 (ft^{-1}). (a) $x-x_k$=0.5 in. (b) $x-x_k$=4.5 in.

FIGURE 1. LONGITUDINAL VELOCITY FLUCTUATIONS DOWNSTREAM OF ROUGHNESS ELEMENT



Mean-velocity distributions downstream of 0.066-in. diam roughness element at $x_b = 2.0$ ft compared with Hartree profiles; ⊙ $U_e/\nu = 1.0 \times 10^6$ (ft^{-1}); △ $U_e/\nu = 1.16 \times 10^6$ (ft^{-1}); ⊡ $U_e/\nu = 1.42 \times 10^6$ (ft^{-1}).

FIGURE 2. MEAN PROFILES IN RECOVERY ZONE

the boundary layer response at this station). Therefore, downstream of $x - x_k = 4.5$ in, the boundary layer is acting like a typical flat plate boundary layer under the influence of a large initial disturbance profile. We will attempt to show similarities between this flowfield (or the forced response to other larger scale perturbations) and the response to acoustical disturbances, by an examination of the results of our computer code.

For the computation of the acoustic response, one must first solve the linear acoustic stability problem (at $\bar{x} = \bar{x}^*$) as described in the previous section. Such analyses have been performed. An eigenvalue search indicates that for a given frequency, the wave numbers corresponding to the eigen solutions occur at propagation velocities on the order of the freestream velocity (actually two to three times larger). Thus the results of Refs. 5 and 6 are correct, even for an acoustical disturbance, in the linear range.

The computer program for the solutions of equations (7) - (10) has been run for certain cases corresponding to the Reynolds number and frequency range of the Spangler and Wells experiments. Figure 3 presents the results of three such computations for the forced response of the boundary layer to acoustic waves of different frequencies (using linear stability theory at $\bar{x}^* = .025$). For these cases the values of $U'_{R.M.S.}$ taken was .3% of the freestream.

Linear stability theory would predict that for the length Reynolds number of interest for the Spangler and Wells experiments ($\approx 3 \times 10^6$ at $x = 12$ ft) a frequency of 27 cycles/sec is within the unstable region, whereas at 150 cycles/sec the flow is stable. This factor is evident in Figure 3. The 43 cycles/sec case exhibits neither amplification or damping (and thus appears neutrally stable) a factor which does not agree with the experimental results which show early transition at 43 cycles. It may be possible that if the computation were carried further in time some amplification would exist.

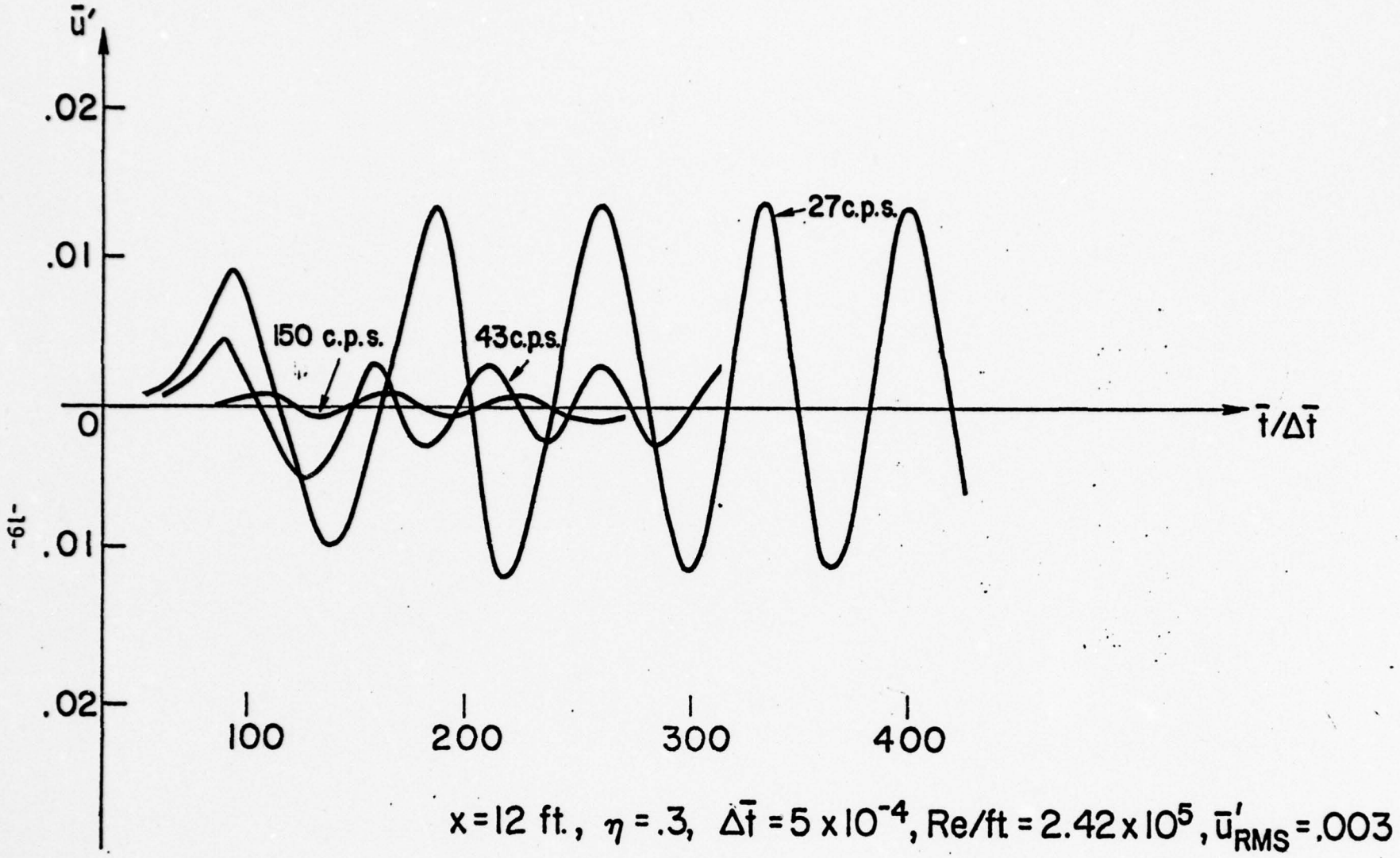


FIGURE 3. FREQUENCY DEPENDENCE ON DAMPING OR AMPLIFICATION OF ACOUSTICAL DISTURBANCE

Figures 4 and 5 present the temporal history, for the 27 cycle/sec case for $U'_{R.M.S.}$ = .3% of freestream at different stations downstream of the leading edge at two different transverse stations. As one moves significantly downstream (i.e., to x = 16 ft) the level of amplification decreases (its maximum is at x = 8 ft, corresponding to an $Re_L = 1.9 \times 10^6$). Such results are in line with previous results, i.e., the amplification curves of Fasel[1] where the amplification increases as one passes into the instability region and decreases as the neutral stability curve is again crossed.

An illustration of severe damping is exhibited in Figure 6, which shows how a large perturbation is damped when the frequency is well outside of stability theory. The exceptionally high value of disturbance is almost completely damped at x = 6 ft.

Much has been said about the effect of acoustics on transition, the application of linear stability theory with the mismatch of propagation velocities between turbulence and acoustic waves, and whether or not Tollmien Schlichting waves are excited under the influence of sound waves. Indeed, the frequency range of the experiments is in line with stability theory. An investigation was initiated to determine what the propagation speed inside the boundary layer was. In all cases, while the disturbances were propagating with the speed of sound along the freestream, the waves within the boundary layer were propagating at a speed of the order of the freestream speed, and thus indeed, the waves are the classical Tollmien Schlichting waves. Such a result is presented in Figure 7. This result is in agreement with the recent experiments reported in Ref. 10.

What is essentially occurring is that the wave propagating along the outer edge has little effect on the boundary layer development. Instead, the major effect is the profile at the leading edge, and thus, the effect of the acoustic wave is only to set up the initial disturbance field (i.e., at $\bar{x} = \bar{x}^*$). Since

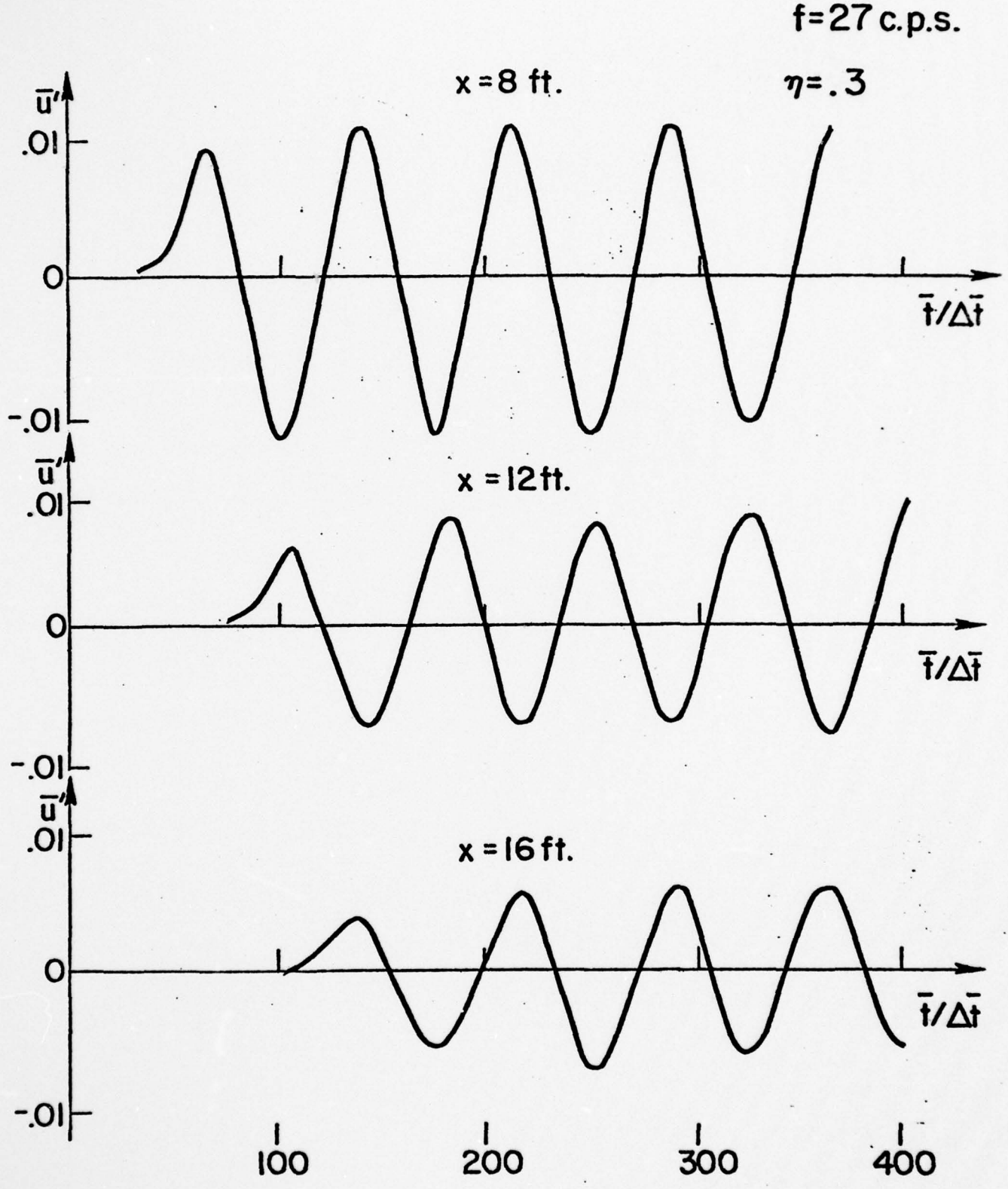


FIGURE 4. TEMPORAL HISTORY DOWNSTREAM OF IMPOSED DISTURBANCE (η = .3, f = 27 cps)

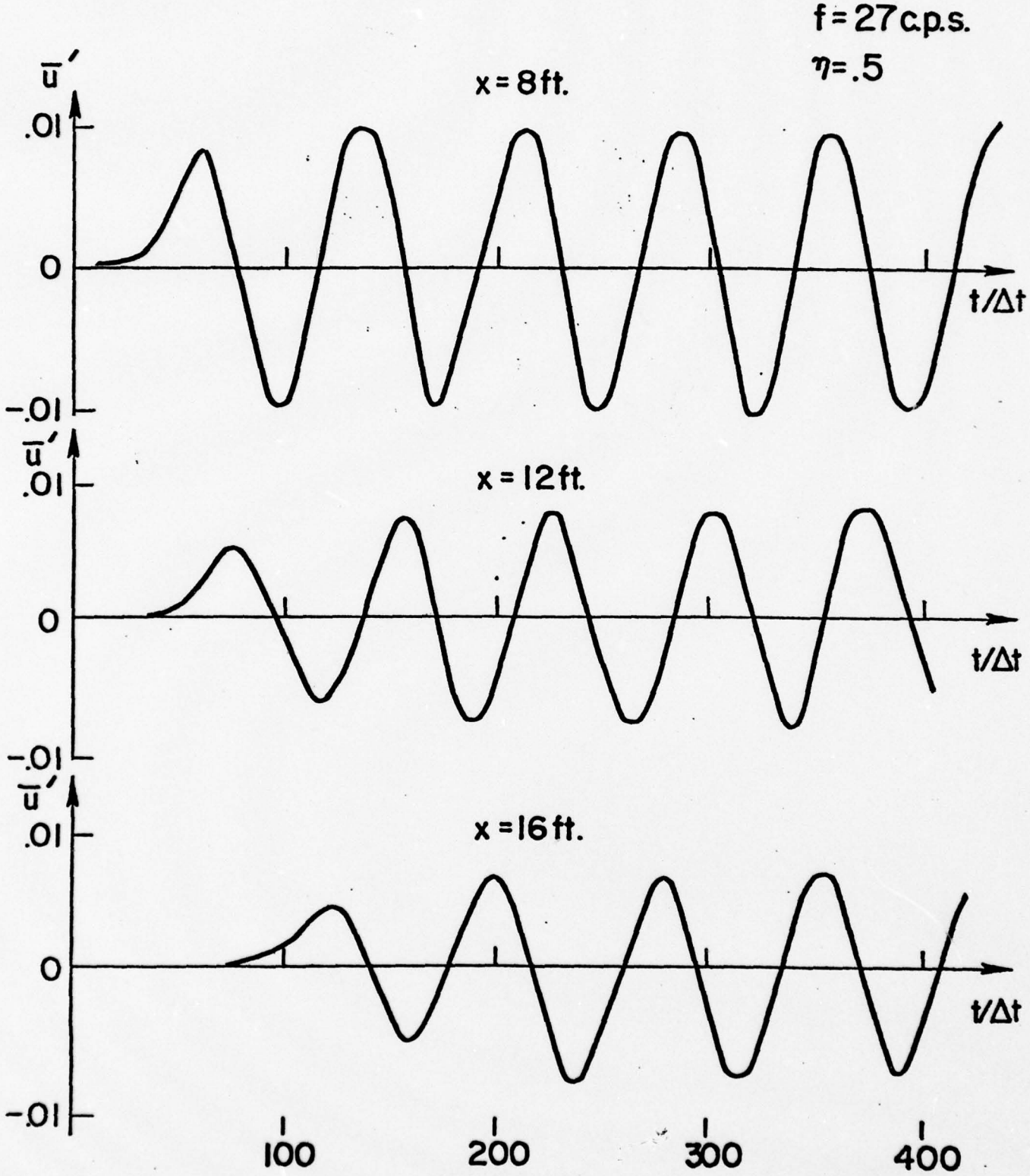


FIGURE 5: TEMPORAL HISTORY DOWNSTREAM OF IMPOSED DISTURBANCE (η = .5, f = 27 cps

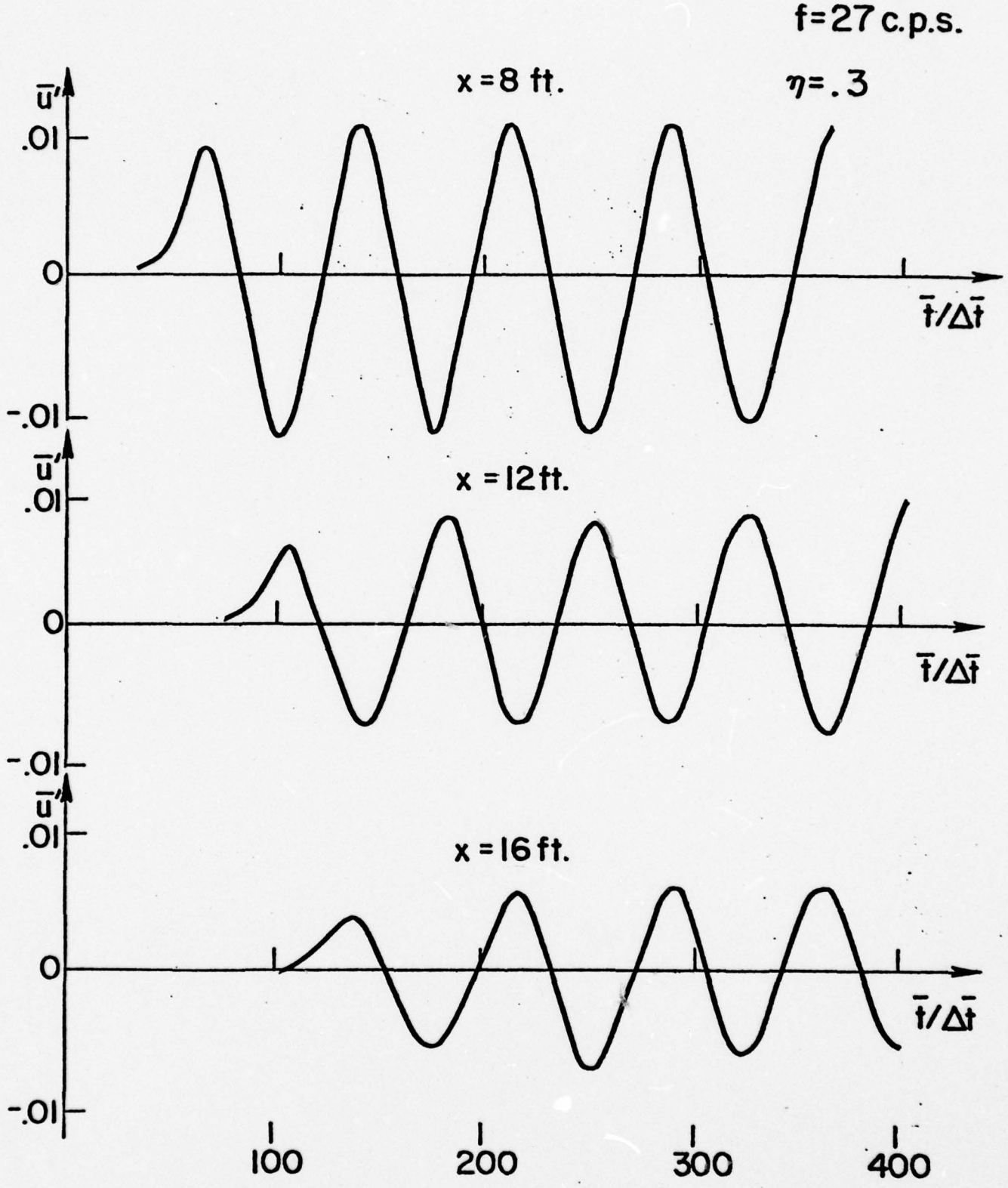


FIGURE 4. TEMPORAL HISTORY DOWNSTREAM OF IMPOSED DISTURBANCE (η = .3, f = 27 cps)

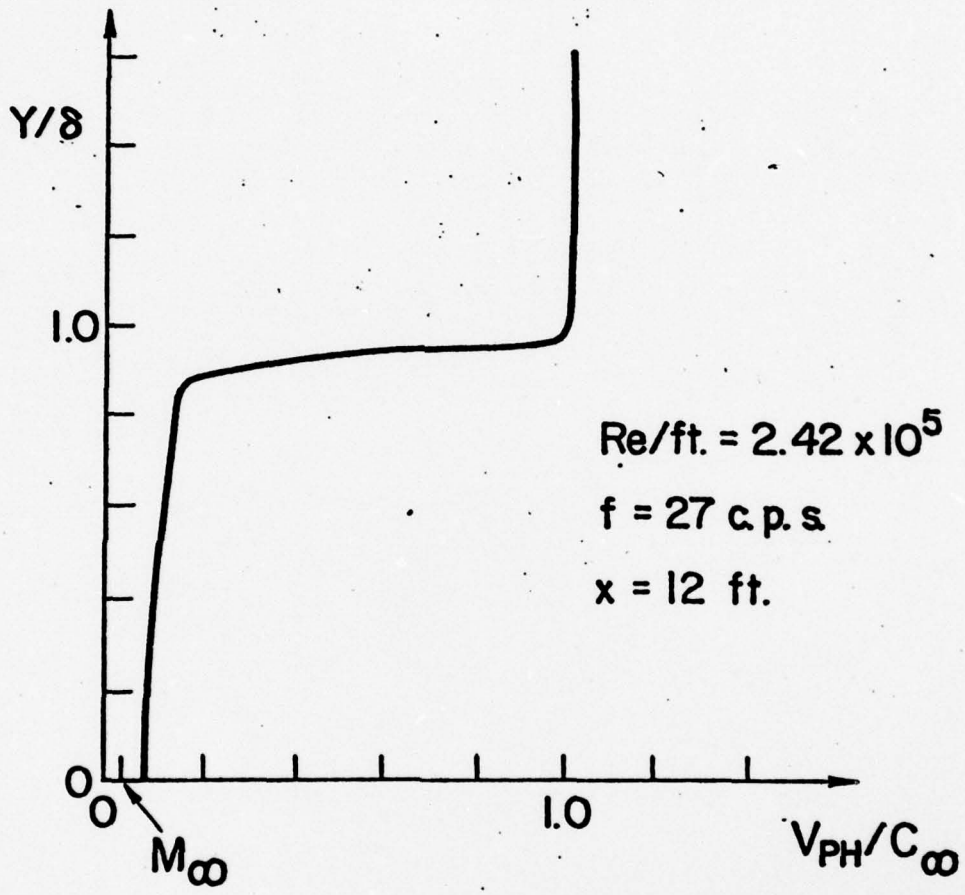


FIGURE 7. PHASE SPEED OF WAVES INSIDE BOUNDARY LAYER UNDER THE INFLUENCE OF ACOUSTICAL DISTURBANCE

we have found that the profiles of References 5 and 6 are representative of the acoustical disturbance profile near the leading edge, the effect of acoustics on transition is, indeed, similar to the effect of other large (or small) perturbations such as the roughness element effects described previously.

In order to test this result, a series of runs were made to compare the results of the present code to those of other investigations. Figures 8 and 9 illustrate how the results of the present analysis compare with the results of Ref. 2. Even though at the outer boundary, waves are propagating at a speed one to two orders of magnitude greater than the freestream velocity, the results of the present investigation with respect to boundary layer response are in good agreement with the results of Ref. 2.

Figure 10 illustrates a test case run to see how the method compares with linear stability theory. For the case $U_\infty/\nu = .6 \times 10^5$/ft and f = 25 cps (with L = 12 ft) linear stability theory predicts that the Reynolds number based on displacement thickness for neutral stability is approximately 1125. For $Re^* < 1125$ waves are damped, for $Re^* > 1125$ waves are amplified. The results of the present calculation indicate this to be the case, again confirming both the utility of the method and the fact that the response of a boundary layer to acoustic vibrations is similar to the response to other disturbance fields.

A remark is necessary with respect to the utility of the present analysis with respect to the problem of transition. There is much question as to the length of the three dimensional region prior to transition. Indeed, if three-dimensional effects predominate over a long distance prior to transition, two dimensional programs will be unable to predict transition accurately. Such a program though, can be of value with respect to determining basic flow structure in regions prior to the onset of transition, as has been demonstrated here.

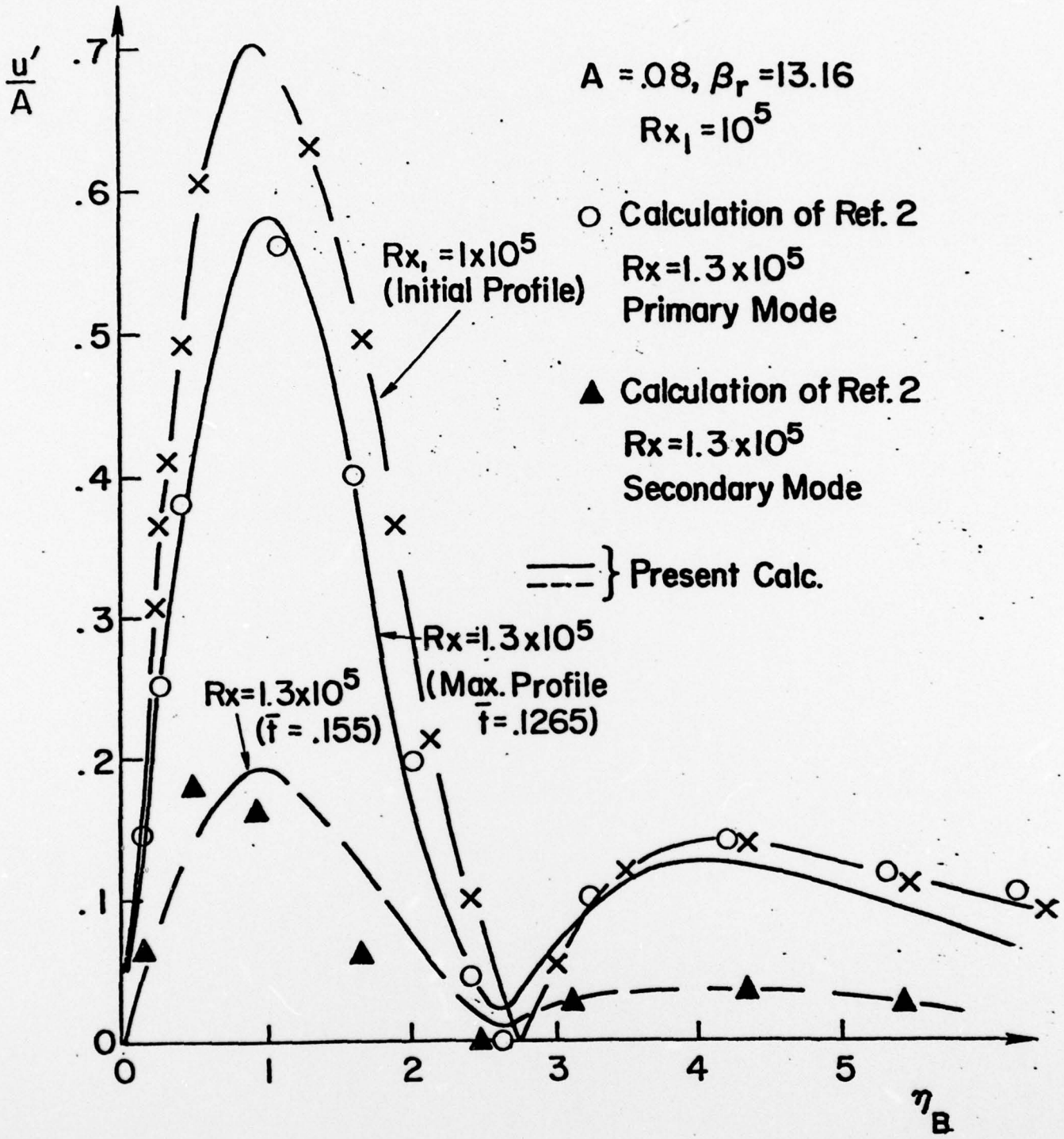


FIGURE 8. COMPARISON OF COMPUTED DISTURBANCE PROFILES WITH DATA OF REFERENCE 2

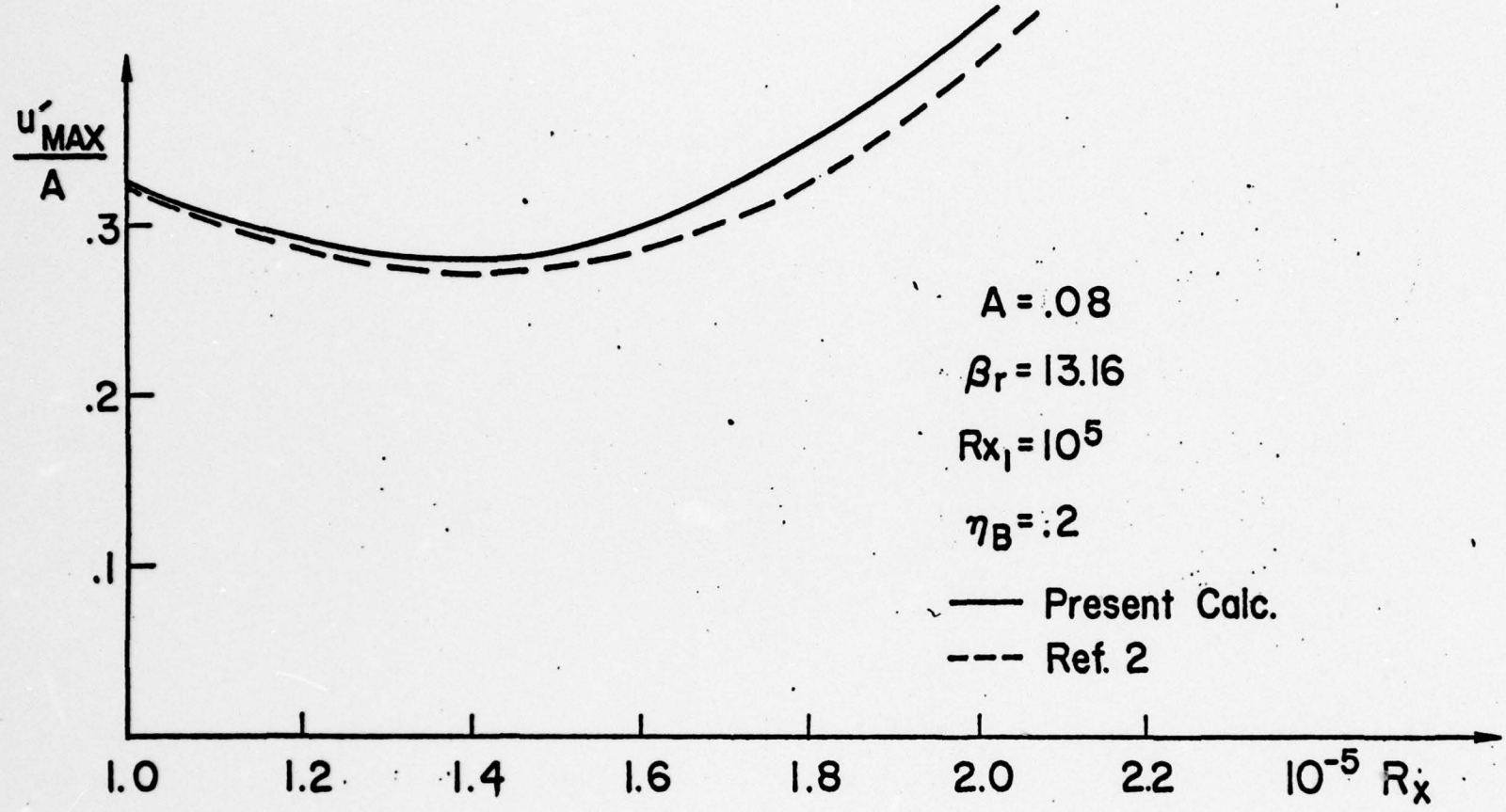


FIGURE 9. COMPARISON OF MAXIMUS DISTURBANCE DISTRIBUTION WITH DATA OF REFERENCE 2

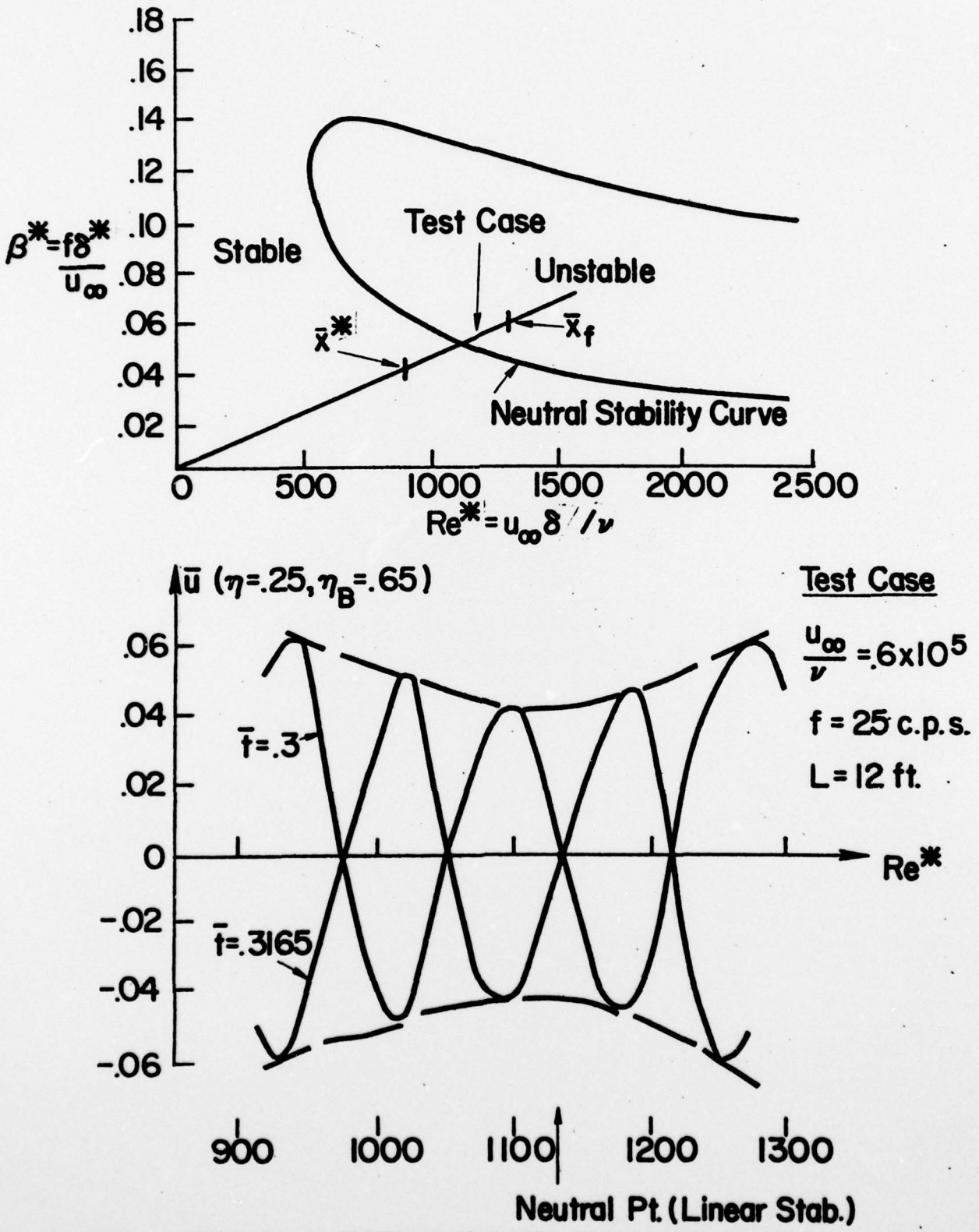


FIGURE 10. COMPARISON OF PROGRAM OUTPUT WITH LINEAR STABILITY THEORY

It is felt that the results achieved to date provide a fundamental understanding with respect to boundary layer response to acoustical disturbances. It seems necessary at this time that the numerical analysis and computer code be modified with respect to the marching step utilized in order to allow the program to have general applicability (for example, to allow the program to be utilized at lower Reynolds numbers). Therefore, in the coming year, the full, compressible, unsteady Navier Stokes equations will be reprogrammed and solved by an explicit MacCormack scheme with time splitting[13]. Thus the small time scale in the normal direction ($\Delta t_y < \Delta y/c$) can be properly accounted for. There are advantages to utilizing an explicit scheme for the complex system to be solved (as opposed to a fully implicit scheme or an A.D.I.). When solving the full compressible Navier Stokes equations, all derivatives in x and y are immediately computed and thus the solution of the full system is no more complex, in principle, than the solution of the boundary layer system. In addition, whereas an implicit scheme allows one to utilize a bigger time step from a stability viewpoint one is still bound by the small time step criterion from the viewpoint of accuracy.

Once the program has been changed to allow for general applicability, the program will be extended so that a wide range of parameters can be varied. The effects of Mach number, type of disturbance, wall temperature, wall suction or injection, and wall compliance will be studied after the program has been properly modified.

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